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Total No. of Questions: 8]

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## **MMPD-104**

## M.E./M.Tech., I Semester

Examination, December 2017

## Theory of Vibration

Time: Three Hours

Maximum Marks: 70

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Note: i) Attempt any five questions.

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- ii) All questions carry equal marks.
- iii) Assume suitable data, if necessary.
- Show that for viscous damping, loss factor is independent of the amplitude and proportional to frequency.
  - A heavy machine of mass m is mounted through a resilient system on a foundation. The resilient system comprises of a spring of stiffness k and a viscous damper with damping coefficient c, the machine produces as excitation force  $F(t)=F_0\sin\omega t$ . Derive the formula for total force transmitted to the foundation and prove that the forcing frequency should be greater than  $\sqrt{2}$  times the natural frequency of the system in order to achieve vibration isolation.
- What is the difference between vibration isolator and vibration absorber?

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[2]

A double pendulum having length l is shown in figure 1. Determine natural frequency of double pendulum when K=50N/m,  $m_1=3kg$ ,  $m_2=5kg$ , l=200mm and a=100mm.

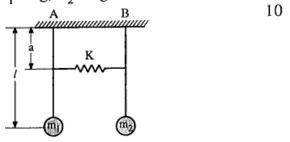
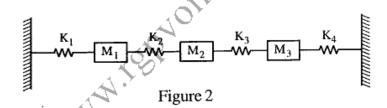
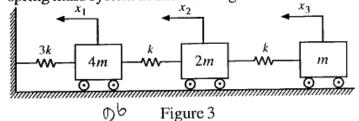


Figure 1

- State and prove Maxwell's Reciprocal theorem. 3.
  - Formulate Eigen value problem for system shown in Figure 2 and then find natural frequencies and mode shapes.  $K_1=2, K_2=3, K_3=4, K_4=10N/m, M_1=2, M_2=3 \text{ and } M_3=10 \text{ kg}$



- A uniform beam fixed at one end and simply supported at the other is having transverse vibrations. Derive the suitable expression for frequency.
  - Define stiffness influence coefficient. Determine the influence coefficient of the three degrees of freedom spring mass system as shown in figure 3.



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5. a)	<ul> <li>a) Use a one element finite element model to approximate the lowest natural frequency and mode shape for a uniform fixed free beam.</li> <li>b) A shaft of negligible weight 6cm diameter and 5 meters long is simply supported at the ends and carries four weights 50kg each at equal distance over the length of</li> </ul>			<ul> <li>b) How are the mean square value, autocorrelation function and power spectral density function of stationary random process related? Discuss.</li> </ul>						
b)					8.	Wr	ite short notes on :		14	
		quency of vibration by Dunkerley's	www.rgpvonline.com	www	a)		Torsionally Equivalent shaft			_
	method. Take $E = 2$	k 106 kg/cm <sup>2</sup> . 7				b)	Jump Phenomenon			WWW
6. a)	problem?  Differentiate between i) Sub harmonic an ii) Hard and soft Sp The non-dimension governing the motion	Differentiate between:  Sub harmonic and Super harmonic oscillations  Hard and soft Spring  The non-dimensional form of the nonlinear equation overning the motion of a pendulum is $\ddot{\theta} + \sin \theta = 0$ Derive the general equation defining the phase plane		www.rgpvonline.com www.r		c)	Perturbation method  ******	*****		www.rgpvonline.com www.r
7. a)	when $\theta = 0$ iii) What is the mapendulum will so  Briefly differentiate  i) Sample space and  ii) Stationary random	between: 7 and an ensemble born and non stationary random process	www.rgpvonline.com	www.rgpvonline.com			THE STATE OF THE S			www.rgpvonline.com
	<ul><li>iii) Deterministic and non deterministic forcing functions</li></ul>						8&			
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